

**Midterm Exam: Feedback Control Systems (ENSC383), Fall 2007**

**School of Engineering Science, Simon Fraser University**

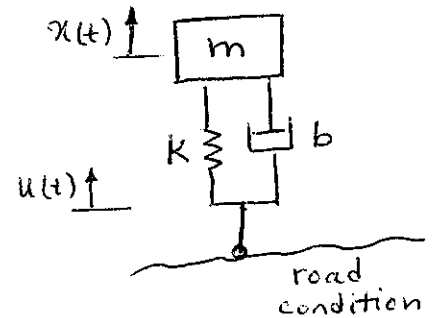
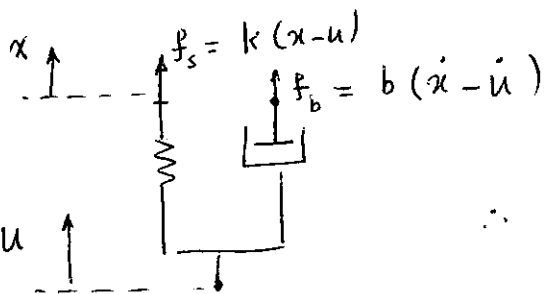
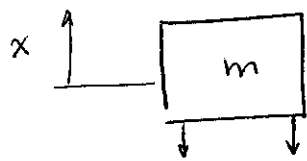
**Note: Questions are marked out of 22. This exam has 22% of the overall mark**

**Name:** \_\_\_\_\_

**Time: 1hr:30min**

1. (5 marks) A simplified car model is given by the figure below, where  $b$  is the damping coefficient,  $m$  is the mass of the car, and  $k$  is the stiffness constant of the springs. Write the dynamic equations of the system and obtain the transfer function from input  $u$  to output  $x$ .

Free body diagram



$$\Rightarrow -k(x-u) - b(\dot{x}-\dot{u}) = m\ddot{x}$$

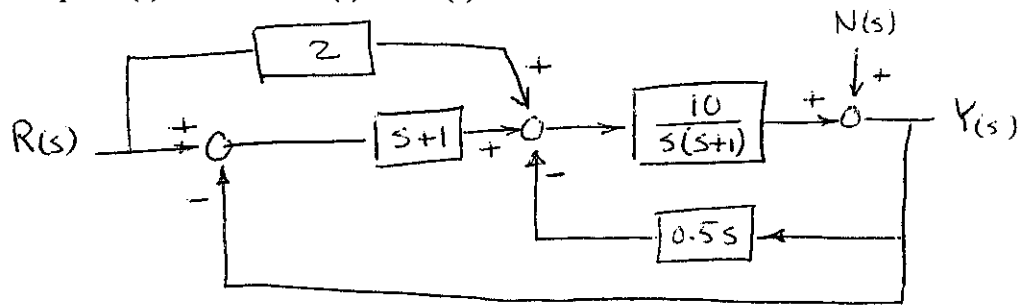
$$\therefore m\ddot{x} + b\dot{x} + kx = b\dot{u} + ku$$

Laplace domain:

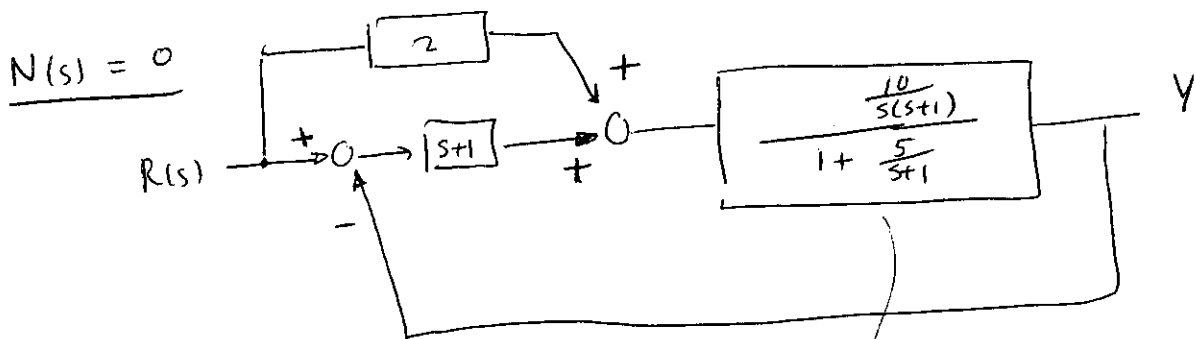
$$(ms^2 + bs + k)X(s) = (bs + k)U(s)$$

$$\frac{X(s)}{U(s)} = \frac{bs + k}{ms^2 + bs + k}$$

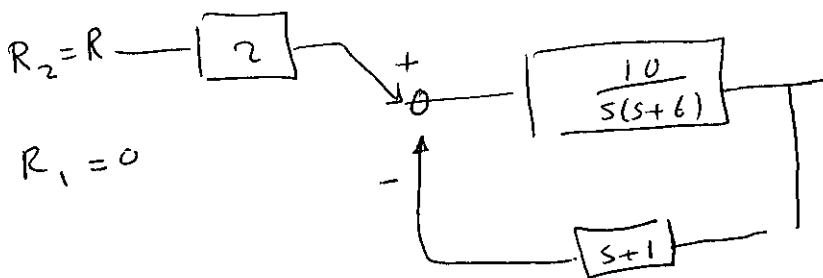
2. (5 marks) The block diagram of a feedback control system is shown in the following figure. Obtain the output  $Y(s)$  in terms of  $R(s)$  and  $N(s)$ .



Let us use superposition  $\rightarrow Y(s) = H_R(s) R(s) + H_N(s) N(s)$



$$\frac{10}{s^2 + s + s} = \frac{10}{s(s+6)}$$



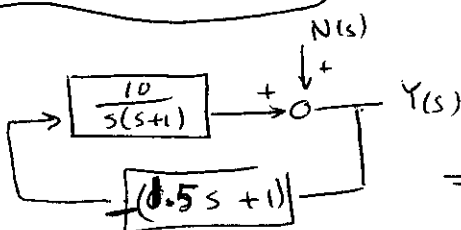
$$Y_{R_2=R} = 2 \frac{\frac{10}{s(s+6)}}{1 + \frac{10(s+1)}{s(s+6)}} = \frac{20}{s^2 + 6s + 10s + 10} = \frac{20}{s^2 + 16s + 10}$$

$R_2 = 0$   
 $R_1 = R$

$$Y_{R_1=R} = \frac{10(s+1)}{1 + \frac{10(s+1)}{s(s+6)}} = \frac{10(s+1)}{s^2 + 6s + 10s + 10} = \frac{10(s+1)}{s^2 + 16s + 10}$$

$$\therefore H_R(s) = \frac{10s + 30}{s^2 + 16s + 10}$$

$R = 0$

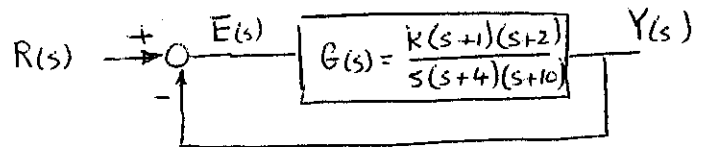


$$\frac{Y(s)}{N(s)} = \frac{1}{1 + \frac{10(0.5s+1)}{s(s+1)}} = \frac{s^2 + s}{s^2 + 16s + 10}$$

Thus  $Y(s) = \frac{10s + 30}{s^2 + 16s + 10} R(s) + \frac{s(s+1)}{s^2 + 16s + 10} N(s)$

3. Consider the unity feedback system shown below.

- a. (4 marks) Find the range of  $k$  for which the closed-loop system is stable  
 b. (3 marks) What is the steady-state error for unit ramp and step inputs.



(a) Closed loop system:

$$\frac{Y(s)}{R(s)} = \frac{\frac{k(s+1)(s+2)}{s(s+4)(s+10)}}{1 + \frac{k(s+1)(s+2)}{s(s+4)(s+10)}} = \frac{k(s+1)(s+2)}{s(s^2+14s+40) + k(s^2+3s+2)}$$

$$s^3 + 14s^2 + 40s + ks^2 + 3ks + 2k = 0 \quad \leftarrow \text{char. eq.}$$

$$s^3 + (14+k)s^2 + (40+3k)s + 2k = 0$$

Routh's Test

|       |                                   |       |
|-------|-----------------------------------|-------|
| $s^3$ | 1                                 | 40+3k |
| $s^2$ | 14+k                              | 2k    |
| $s^1$ | $\frac{(14+k)(40+3k) - 2k}{14+k}$ |       |
| $s^0$ | 2k                                |       |

$$14+k > 0 \rightarrow k > -14$$

$$2k > 0 \rightarrow k > 0$$

$$\frac{(14+k)(40+3k) - 2k}{14+k} > 0 \rightarrow 14 \times 40 + 2k + 40k + 3k^2 - 2k > 0$$

$$3k^2 + 80k + 560 > 0$$

$$k_1, k_2 = \frac{-40 \pm \sqrt{1600 - 3 \times 560}}{3}$$

Complex roots

Thus always positive

Hence  $k > 0$

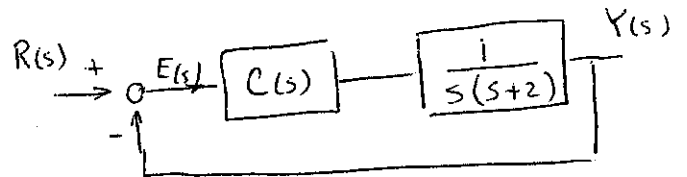
(b) ~~Unit step~~  $\Rightarrow \frac{E(s)}{R(s)} = \frac{1}{1 + \frac{k(s+1)(s+2)}{s(s+4)(s+10)}} = \frac{s(s+4)(s+10)}{s(s+4)(s+10) + k(s+1)(s+2)}$

$$R(s) = \frac{1}{s^2} \Rightarrow E(s) = \frac{s(s+4)(s+10)}{s[s(s+4)(s+10) + k(s+1)(s+2)]}$$

$$e(t) = \lim_{s \rightarrow 0} s E(s) = \frac{4 \times 10}{k \times 2} = \frac{20}{k}$$

$$R(s) = \frac{1}{s} \Rightarrow E(s) = 0$$

4. (5 marks) In the feedback system below, the desired system response to a step input is specified as peak time  $t_p = 1$  sec and overshoot  $M_p = 5\%$ . Determine whether both specifications can be met simultaneously by selecting  $C(s)$  as a proportional controller and a proportional-derivative controller. Obtain the corresponding controllers.



$C(s)$ : Proportional controller, i.e.,  $C(s) = k$

$$\therefore \frac{Y}{R} = \frac{\frac{k}{s(s+2)}}{1 + \frac{k}{s(s+2)}} = \frac{k}{s^2 + 2s + k}$$

$$\omega_n^2 = k$$

$$2\zeta\omega_n = 2 \Rightarrow \zeta = \frac{1}{\omega_n}$$

$$\omega_n = \sqrt{k}$$

$$2\zeta\sqrt{k} = 2 \Rightarrow \zeta = \frac{1}{\sqrt{k}}$$

$$\begin{aligned} \omega_n &= \sqrt{k} \\ \zeta &= \frac{1}{\sqrt{k}} \\ k &= \frac{1}{\zeta^2} \end{aligned}$$

$$\begin{cases} t_p = 1 = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} \Rightarrow \omega_n = \frac{\pi}{\sqrt{1-\zeta^2}} \\ M_p = e^{-\frac{\zeta\pi}{\sqrt{1-\zeta^2}}} = 0.05 \Rightarrow -\frac{\zeta\pi}{\sqrt{1-\zeta^2}} = \ln(0.05) \end{cases}$$

$$\frac{\zeta^2 \pi^2}{1-\zeta^2} = (\ln(0.05))^2 = 8.9744$$

$$\zeta^2 = 0.9093(1-\zeta^2)$$

$$(1+0.9093)\zeta^2 = 0.9093$$

$$\zeta^2 = 0.476 \Rightarrow \zeta = 0.6899$$

$$\omega_n = \frac{\pi}{\sqrt{1-\zeta^2}} = 4.3399$$

$$k = 18.83$$

$$C(s) = k_p + k_d s \Rightarrow \frac{Y}{R} = \frac{(k_p + k_d s)}{s(s+2)} = \frac{k_p + k_d s}{s^2 + (2+k_d)s + k_p}$$

$$\text{④} \& \text{⑤} \Rightarrow k_p = \omega_n^2 = 18.83$$

$$2+k_d = 2\zeta\omega_n \Rightarrow 2+k_d = 2 \times 0.6899 \times 4.3399$$

$$k_d = 3.9884$$

$\Rightarrow$  Can meet spec's

Check zero:  $\frac{1}{2}$  poles

$$s^2 + 5.9884s + 18.83 = 0$$

$$s_{1,2} = \frac{-5.9884 \pm \sqrt{5.9884^2 - 4 \times 18.83}}{2 \times 1}$$

$$= -2.994 \pm j3.14$$

$$\text{Zero} \rightarrow -\frac{k_p}{k_d} = -\frac{18.83}{3.9884} = -4.7$$